

Introducció al Mètode dels Elements Finit

Elements de barra

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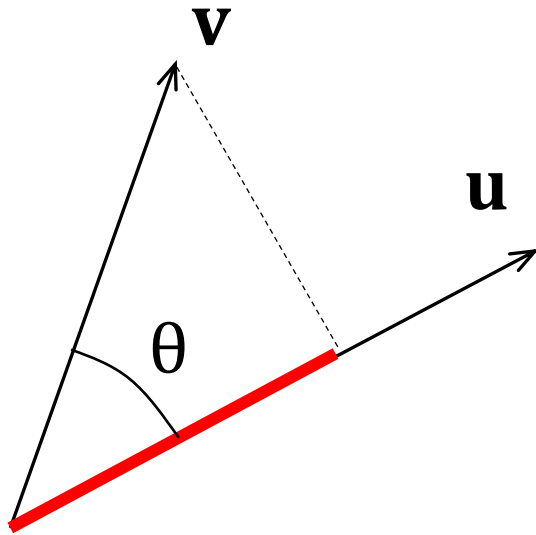
UNIVERSITAT POLITÈCNICA
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Teoria de l'elasticitat

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Elements de barra

Mètode dels Residus Ponderats



$$\mathbf{u} \cdot \mathbf{v} = u_i v_i = c$$

$$\mathbf{u} \cdot \mathbf{v} = \|\mathbf{u}\| \|\mathbf{v}\| \cos \theta$$

Projecció d' $\mathbf{u}$ a $\mathbf{v}$, o vice-versa

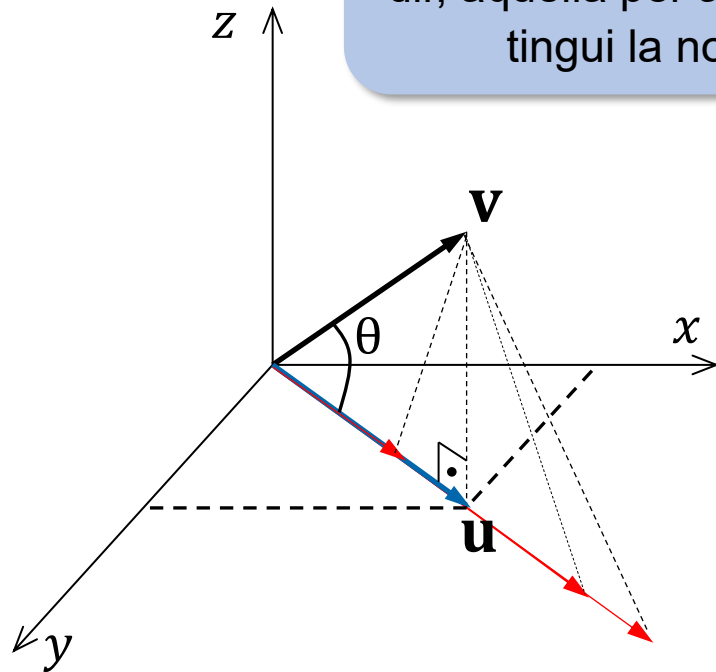
Dos vectors son ortogonals si:

$$\mathbf{u} \cdot \mathbf{v} = \|\mathbf{u}\| \|\mathbf{v}\| \cos 0 \rightarrow \mathbf{u} \cdot \mathbf{v} = 0$$

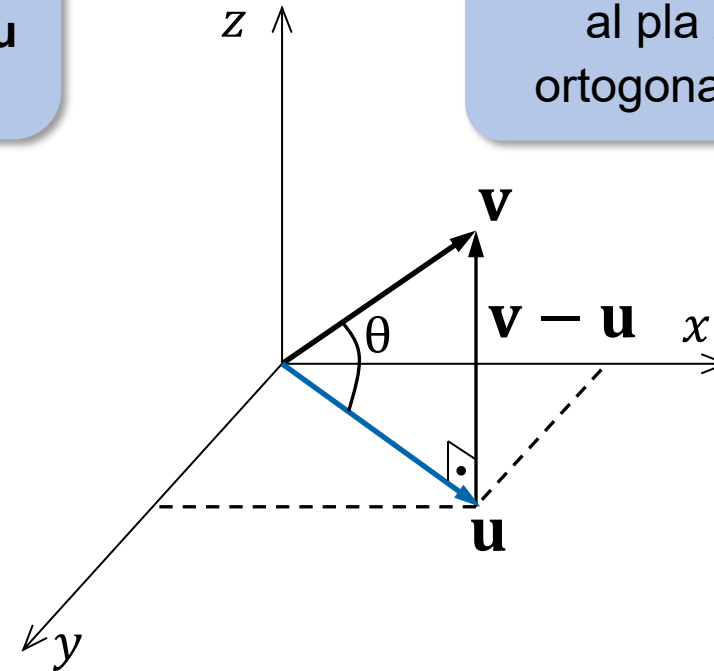
Exemple:

Volem trobar la millor aproximació de $\mathbf{v}(x,y,z)$ en el pla xy :

La millor aproximació serà aquella que condueixi a l'error més petit, és a dir, aquella per a la qual el vector $\mathbf{v}-\mathbf{u}$ tingui la norma més petita



La norma del vector $(\mathbf{v} - \mathbf{u})$ és mínima quan aquest és ortogonal al pla xy , és a dir, quan és ortogonal al vector aproximat $\mathbf{u}$



$$\mathbf{u} \cdot (\mathbf{v} - \mathbf{u}) = 0$$

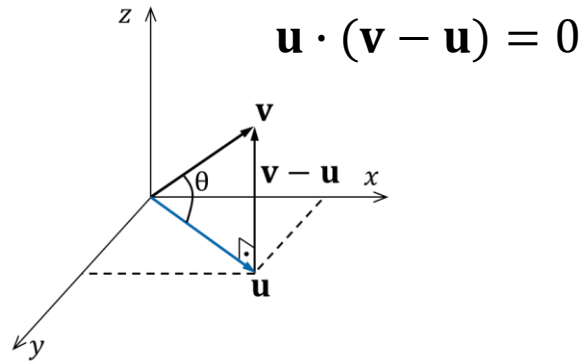
Podem generalitzar el concepte de producte intern de vectors per les funcions:

$$\mathbf{u} \cdot \mathbf{v} = \sum u_i v_i \quad \longrightarrow \quad \langle f, g \rangle = \iint_D f(x, y) g(x, y) dx dy$$

Dues funcions f i g són ortogonals, en un interval, si el producte intern entre elles és nul:

$$\iint_D f(x, y) g(x, y) dx dy = 0$$

En el contexte vectorial



En el contexte de funcions:

$$\iint_D f(x, y)g(x, y)dxdy = 0$$



$$\iint_D R(a_0, a_1, \dots, a_n, x, y)\phi_j(x, y) = 0$$

En el Mètode de Galerkin, l'objectiu és minimitzar la funció residu:

Funció d'aproximació

$$u_a = u_0(x, y) + \sum_{j=1}^N a_j \phi_j(x, y)$$

Funció residu

=

Error d'aproximació

$$R(a_0, a_1, \dots, a_n, x, y) = L(u_a)$$



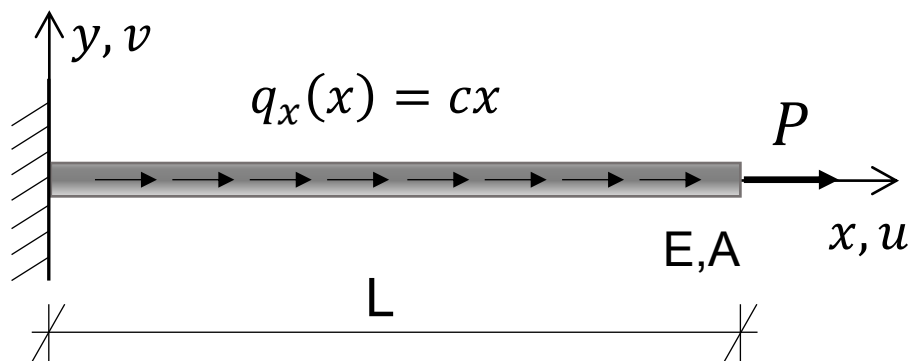
La funció d'aproximació ha de ser ortogonal a l'error

Elements de barra



Aplicació a una barra

Equilibri en una barra



Equació diferencial:

$$EA \frac{d^2 u_0(x)}{dx^2} = -q_x(x)$$

Condicions de contorn:

$$\left\{ \begin{array}{l} u_0(0) = 0 \end{array} \right.$$

Essencial

$$\left\{ \begin{array}{l} N(L) = P \rightarrow \frac{du_0(L)}{dx} EA = P \end{array} \right.$$

Natural

Funció d'aproximació:

$$\hat{u}(x) = \sum_{i=0}^n a_i x^i$$

Funció resídu:

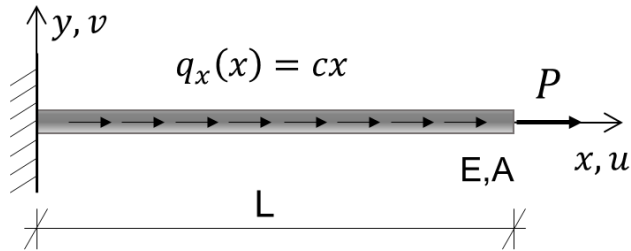
$$R(\hat{u}(x)) = EA \frac{d^2 \hat{u}(x)}{dx^2} + q_x(x)$$

Aplicació del MRP:

$$\int_0^L R(\hat{u}(x)) W_i dx = 0$$

$$\int_0^L \left(EA \frac{d^2 \hat{u}(x)}{dx^2} + q_x(x) \right) W_i dx = 0$$

Equilibri en una barra – Forma integral



$$\int_0^L \left(EA \frac{d^2 \hat{u}(x)}{dx^2} + q_x \right) W_i dx = 0 \quad \Rightarrow \quad \int_0^L \left(EA \frac{d^2 \hat{u}(x)}{dx^2} \right) W_i dx + \int_0^L (q_x) W_i dx = 0$$

Aplicant l'integració per parts: $\int u dv = uv - \int v du$

$$\int_0^L EA \frac{d^2 \hat{u}(x)}{dx^2} W_i dx = \left[W_i \frac{d\hat{u}(x)}{dx} EA \right] \Big|_0^L - \int_0^L EA \frac{d\hat{u}(x)}{dx} \frac{dW_i}{dx} dx$$

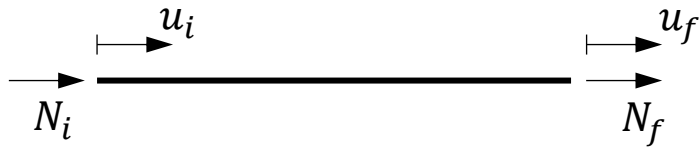
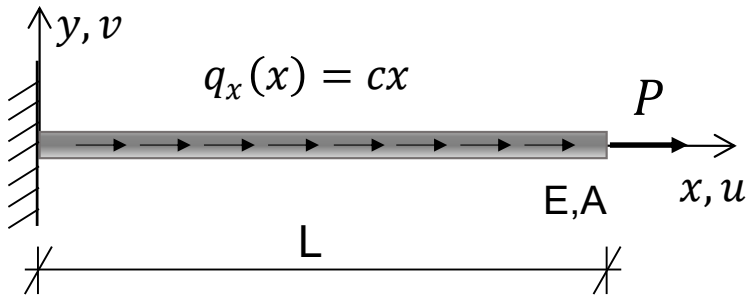
$$\int_0^L \left(EA \frac{d^2 \hat{u}(x)}{dx^2} + q_x \right) W_i dx = \left[W_i \frac{d\hat{u}(x)}{dx} EA \right] \Big|_0^L - \int_0^L EA \frac{d\hat{u}(x)}{dx} \frac{dW_i}{dx} dx + \int_0^L (q_x) W_i dx$$

$$\int_0^L \left(\frac{d^2 \hat{u}(x)}{dx^2} + \frac{c x}{EA} \right) W_i dx = W_i(L) \frac{d\hat{u}(L)}{dx} EA - W_i(0) \frac{d\hat{u}(0)}{dx} EA - \int_0^L EA \frac{d\hat{u}(x)}{dx} \frac{dW_i}{dx} dx + \int_0^L q_x W_i dx$$

Condició de contorn natural:

Apareixen naturalment en el procés d'integració per parts





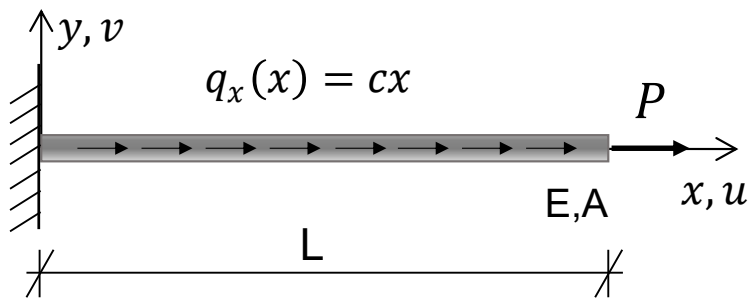
$$W_i(L) \frac{d\hat{u}(L)}{dx} EA - W_i(0) \frac{d\hat{u}(0)}{dx} EA - \int_0^L EA \frac{d\hat{u}(x)}{dx} \frac{dW_i}{dx} dx + \int_0^L q_x W_i dx = 0$$

$$N(x) = \frac{d\hat{u}(x)}{dx} EA$$



$$W_i(L) N_f + W_i(0) N_i - \int_0^L EA \frac{d\hat{u}(x)}{dx} \frac{dW_i}{dx} dx + \int_0^L q_x W_i dx = 0$$

Aquesta equació significa, físicament, una imposició d'equilibri en forma integral per a un medi unidimensional amb comportament axial.



$$W_i(L) N_f + W_i(0) N_i - \int_0^L EA \frac{d\hat{u}(x)}{dx} \frac{dW_i}{dx} dx + \int_0^L q_x W_i dx = 0$$

La funció d'aproximació $\hat{u}(x)$ ha de satisfer la condició de contorn essencial del problema:

$$u_a = u_0(x) + \sum_{i=1}^N a_i \phi_i(x) \xrightarrow[\phi_i(x) = x^i]{u_0(x) = a_0} \hat{u}(x) = \sum_{i=0}^n a_i x^i \xrightarrow[a_0 = 0]{\hat{u}(0) = 0} \hat{u}(x) = \sum_{i=1}^n a_i x^i$$

Adoptant $n=2$: $\hat{u}(x) = a_1 x + a_2 x^2$

Pel Mètode dels Residus Ponderats:

$$\int_0^L R(\hat{u}(x)) W_i dx = 0$$



Versió de Galerkin del mètode:

$$W_i(x) = \phi_i(x) = x^i$$

Les funcions de pes són les mateixes que les utilitzades en la funció d'aproximació

$$W_i(L) N_f + W_i(0) N_i - \int_0^L EA \frac{d\hat{u}(x)}{dx} \frac{dW_i}{dx} dx + \int_0^L q_x W_i dx = 0$$

$$\hat{u}(x) = a_1 x + a_2 x^2$$



$$W_1(x) = x$$

$$W_2(x) = x^2$$



$$dW_1(x)/dx = 1$$

$$dW_2(x)/dx = 2x$$

$$i = 1: \quad L \cdot P - 0 \cdot (a_1 + 2a_2 L) \cdot EA - \int_0^L EA \cdot (a_1 + 2a_2 x) \cdot 1 dx + \int_0^L cx \cdot x dx = 0$$

$$L \cdot P - EA \cdot (a_1 L + a_2 L^2) + \frac{c L^3}{3} = 0$$

$$i = 2: \quad L^2 \cdot P - 0 \cdot (a_1 + 2a_2 L) \cdot EA - \int_0^L EA \cdot (a_1 + 2a_2 x) \cdot 2x dx + \int_0^L cx \cdot x^2 dx = 0$$

$$L^2 \cdot P - EA \cdot \left(a_1 L^2 + a_2 \frac{4L^3}{3} \right) + \frac{c L^4}{4} = 0$$

Example

$$\begin{cases} L \cdot P - EA \cdot (a_1 L + a_2 L^2) + \frac{c L^3}{3} = 0 \\ L^2 \cdot P - EA \cdot \left(a_1 L^2 + a_2 \frac{4L^3}{3} \right) + \frac{c L^4}{4} = 0 \end{cases}$$



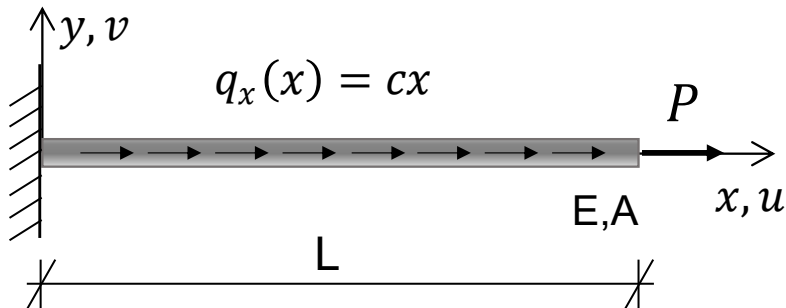
$$\begin{cases} a_1 L + a_2 L^2 = \frac{PL}{EA} + \frac{c L^3}{3EA} \\ a_1 L^2 + a_2 \frac{4L^3}{3} = \frac{PL^2}{EA} + \frac{c L^4}{4EA} \end{cases}$$



$$\begin{cases} a_1 + a_2 L = \frac{P}{EA} + \frac{c L^2}{3EA} \\ a_1 + a_2 \frac{4L}{3} = \frac{P}{EA} + \frac{c L^2}{4EA} \end{cases}$$

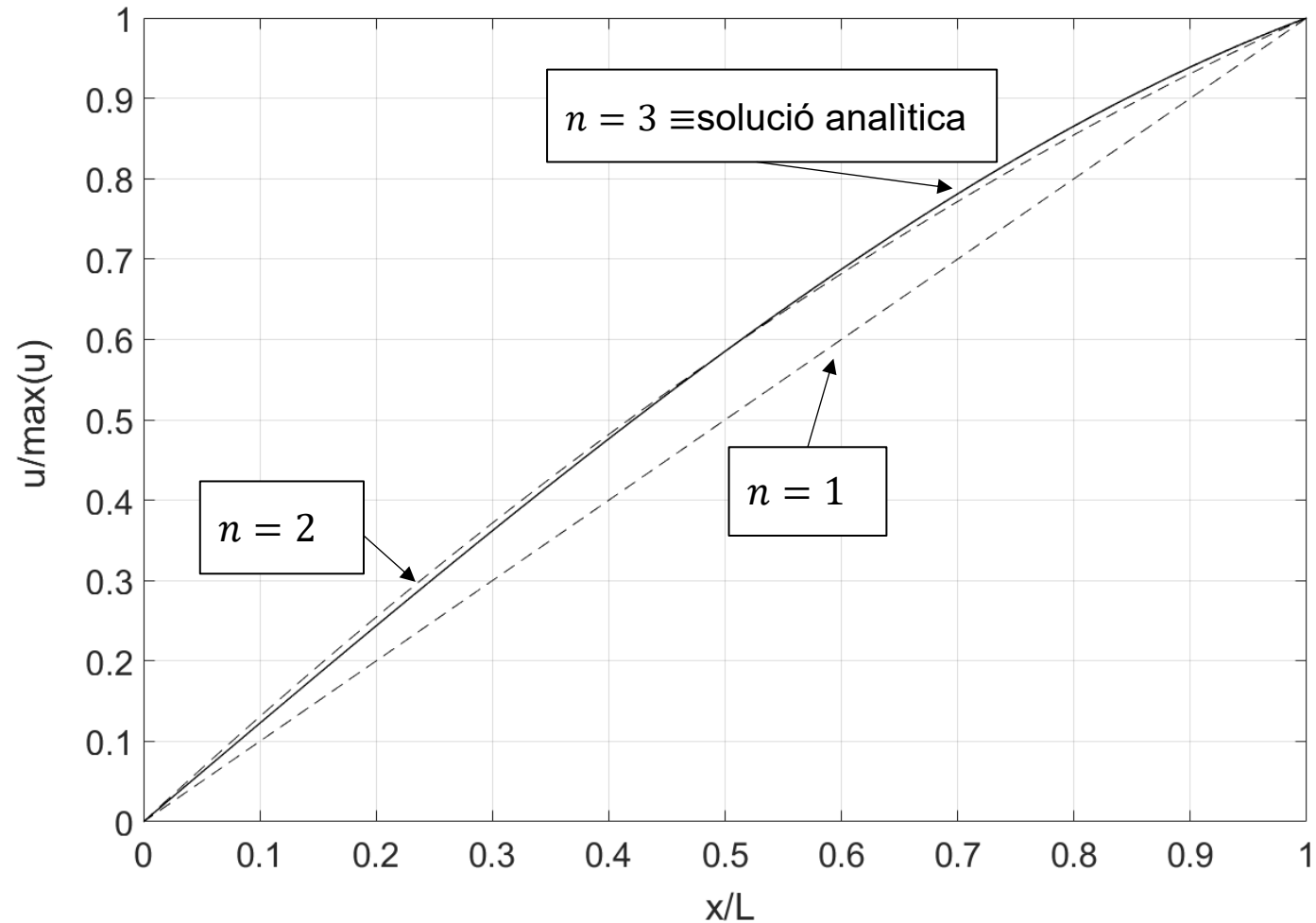


$$\begin{aligned} a_1 &= \frac{P}{EA} + \frac{7c L^2}{12EA} \\ a_2 &= -\frac{c L}{4EA} \end{aligned}$$



$$\hat{u}(x) = \left(\frac{P}{EA} + \frac{7c L^2}{12EA} \right) x - \frac{c L}{4EA} x^2$$

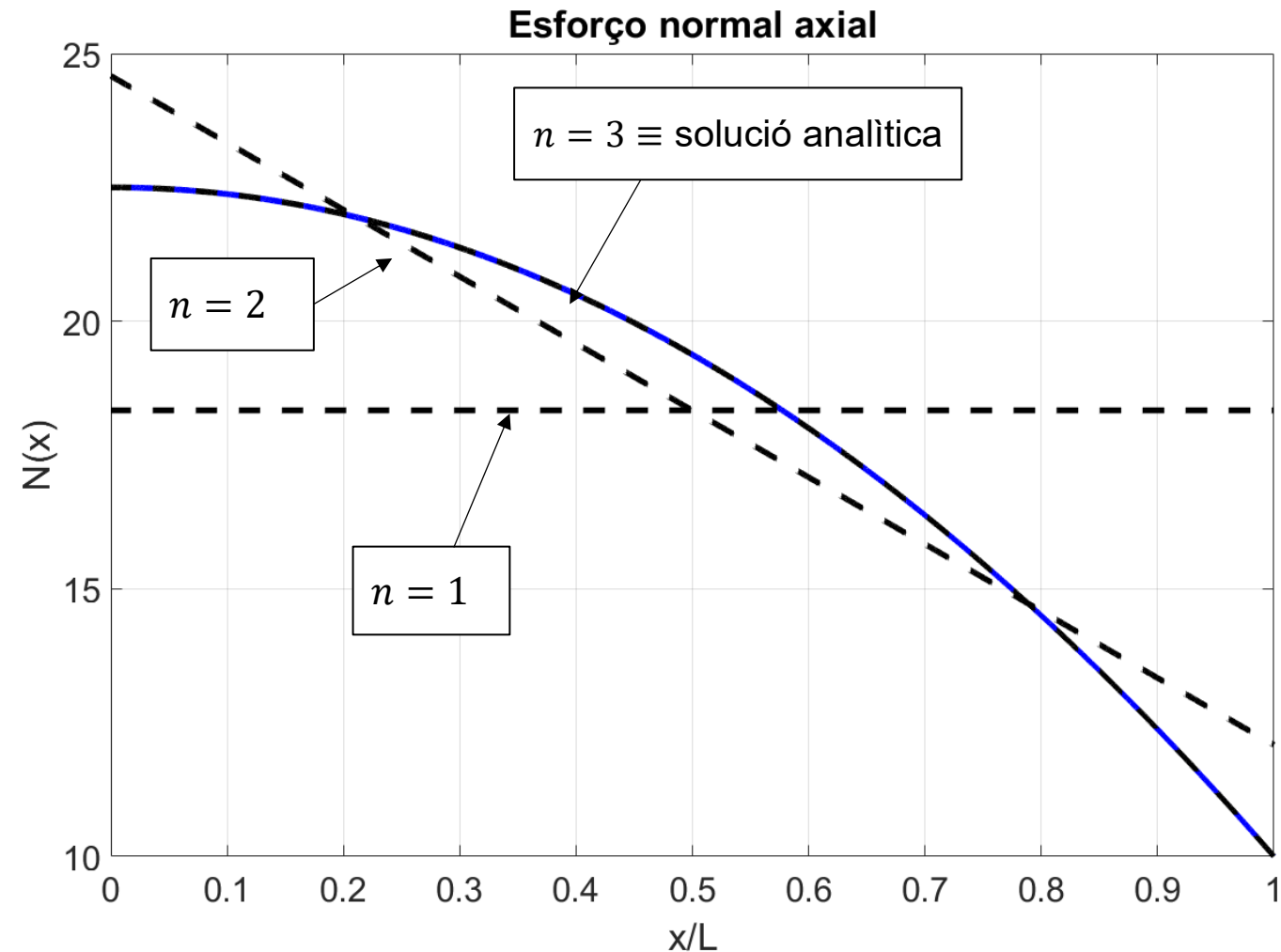
$$\hat{u}(x) = \sum_{i=1}^n a_i x^i$$



Esforç normal vs. funció d'aproximació

$$\hat{u}(x) = \sum_{i=1}^n a_i x^i$$

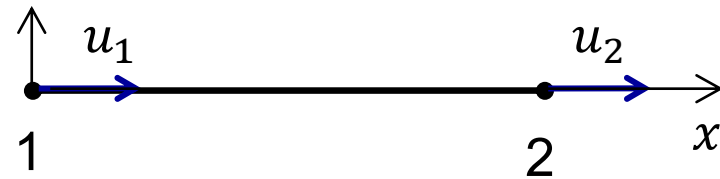
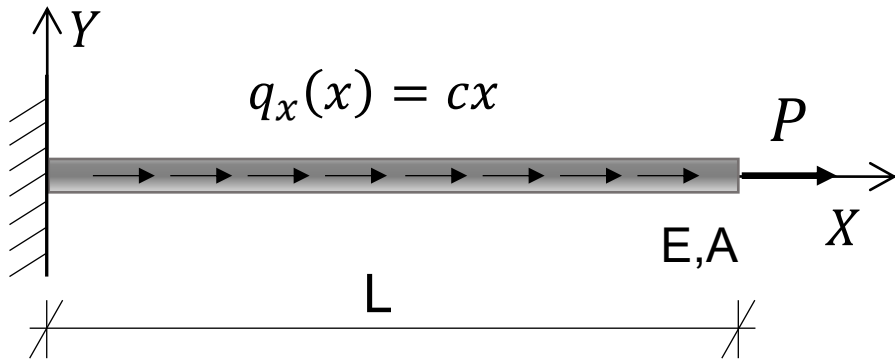
O grau de precisão de grandezas que envolvem derivadas da função de aproximação (esforço normal) é menor que o grau de precisão da função de aproximação fundamental;



Elements de barra

Formulació d'un element de barra

Aproximació del camp de desplaçaments



$$\int_0^L EA \frac{du(x)}{dx} \frac{d\delta u(x)}{dx} dx = \int_0^L q_x \delta u(x) dx + \delta u(L) N_f + \delta u(0) N_i$$

$$\hat{u}(x) = \sum_{i=0}^n a_i x^i \quad \xrightarrow{n=1} \quad \hat{u}(x) = a_0 + a_1 x$$

Sabem:

$$\hat{u}(0) = u_1$$

$$\hat{u}(L) = u_2$$

$$a_0 = u_1$$

$$a_0 + a_1 L = u_2$$

$$a_0 = u_1$$

$$a_1 = \frac{u_2 - u_1}{L}$$

$$\hat{u}(x) = u_1 + \frac{u_2 - u_1}{L} x$$



$$\hat{u}(x) = u_1 + \frac{u_2 - u_1}{L} x$$

$$\hat{u}(x) = \left(1 - \frac{x}{L}\right) u_1 + \frac{x}{L} u_2$$

$$\hat{u}(x) = \begin{bmatrix} 1 - \frac{x}{L} & \frac{x}{L} \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \end{Bmatrix}$$

$$\hat{u}(x) = \mathbf{N}(x) \mathbf{u}$$

$$\frac{d\hat{u}(x)}{dx} = \varepsilon_x(x) = ?$$

$$\varepsilon_x(x) = \frac{d}{dx} \left(\left(1 - \frac{x}{L}\right) u_1 + \frac{x}{L} u_2 \right)$$

$$\varepsilon_x(x) = -\frac{1}{L} u_1 + \frac{1}{L} u_2$$

$$\varepsilon_x(x) = \begin{bmatrix} -\frac{1}{L} & \frac{1}{L} \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \end{Bmatrix} \quad \rightarrow \quad \varepsilon_x(x) = \mathbf{B}(x) \mathbf{u}$$

$$\int_0^L EA \frac{du(x)}{dx} \frac{d\delta u(x)}{dx} dx = \int_0^L q_x \delta u(x) dx + \delta u(L) N_f + \delta u(0) N_i$$

$$u(x) = \mathbf{N}(x) \mathbf{u}$$

$$\varepsilon_x(x) = \mathbf{B}(x) \mathbf{u}$$

$$\delta u(x) = ?$$

Les funcions de pes són les mateixes que les utilitzades en la funció d'aproximació

$$\delta u(x) = \left(1 - \frac{x}{L}\right) \delta u_1 + \frac{x}{L} \delta u_2$$

$$\delta u(x) = \mathbf{N}(x) \delta \mathbf{u}$$

$$\frac{d\delta u(x)}{dx} = \delta \varepsilon_x(x) = -\frac{1}{L} \delta u_1 + \frac{1}{L} \delta u_2 \quad \Rightarrow \quad \delta \varepsilon_x(x) = \mathbf{B}(x) \delta \mathbf{u}$$

$$\int_0^L EA \frac{d\delta u(x)}{dx} \frac{du(x)}{dx} dx = \int_0^L \delta u(x) q_x dx + \delta u(L) N_f + \delta u(0) N_i$$

$$u(x) = \mathbf{N}(x) \mathbf{u}$$

$$\varepsilon_x(x) = \mathbf{B}(x) \mathbf{u}$$

$$\delta u(x) = \mathbf{N}(x) \delta \mathbf{u}$$

$$\delta \varepsilon_x(x) = \mathbf{B}(x) \delta \mathbf{u}$$

$$\int_0^L EA (\mathbf{B}(x) \delta \mathbf{u}) \cdot (\mathbf{B}(x) \mathbf{u}) dx = \int_0^L (\mathbf{N}(x) \delta \mathbf{u}) q_x dx + (\mathbf{N}(L) \delta \mathbf{u}) N_f + (\mathbf{N}(0) \delta \mathbf{u}) N_i$$

$$\int_0^L EA (\mathbf{B}(x) \delta \mathbf{u})^\top (\mathbf{B}(x) \mathbf{u}) dx = \int_0^L (\mathbf{N}(x) \delta \mathbf{u}) q_x dx + (\mathbf{N}(L) \delta \mathbf{u}) N_f + (\mathbf{N}(0) \delta \mathbf{u}) N_i$$

$$\int_0^L \delta \mathbf{u}^\top EA (\mathbf{B}(x))^\top \mathbf{B}(x) \mathbf{u} dx = \int_0^L (\mathbf{N}(x) \delta \mathbf{u}) q_x dx + (\mathbf{N}(L) \delta \mathbf{u}) N_f + (\mathbf{N}(0) \delta \mathbf{u}) N_i$$

$$\int_0^L \delta \mathbf{u}^\top (\mathbf{B}(x))^\top E A \mathbf{B}(x) \mathbf{u} \, dx = \int_0^L (\mathbf{N}(x) \delta \mathbf{u}) q_x \, dx + (\mathbf{N}(L) \delta \mathbf{u}) N_f + (\mathbf{N}(0) \delta \mathbf{u}) N_i$$



Treient $\delta \mathbf{u}$ com a factor comú

$$\delta \mathbf{u}^\top \int_0^L (\mathbf{B}(x))^\top E A \mathbf{B}(x) \, dx \mathbf{u} = \int_0^L (\mathbf{N}(x) \delta \mathbf{u}) q_x \, dx + (\mathbf{N}(L) \delta \mathbf{u}) N_f + (\mathbf{N}(0) \delta \mathbf{u}) N_i$$



$$\delta \mathbf{u}^\top \int_0^L (\mathbf{B}(x))^\top E A \mathbf{B}(x) \, dx \mathbf{u} = \delta \mathbf{u}^\top \int_0^L (\mathbf{N}(x))^\top q_x \, dx + \delta \mathbf{u}^\top (\mathbf{N}(L))^\top N_f + \delta \mathbf{u}^\top (\mathbf{N}(0))^\top N_i$$



$$\delta \mathbf{u}^\top \left(\int_0^L (\mathbf{B}(x))^\top E A \mathbf{B}(x) \, dx \mathbf{u} - \int_0^L (\mathbf{N}(x))^\top q_x \, dx - (\mathbf{N}(L))^\top N_f - (\mathbf{N}(0))^\top N_i \right) = 0$$

$$\delta \mathbf{u}^\top \left(\int_0^L (\mathbf{B}(x))^\top EA \mathbf{B}(x) dx \mathbf{u} - \int_0^L (\mathbf{N}(x))^\top q_x dx - (\mathbf{N}(L))^\top N_f - (\mathbf{N}(0))^\top N_i \right) = 0$$



Per $\delta \mathbf{u} \neq \mathbf{0}$

$$\int_0^L (\mathbf{B}(x))^\top EA \mathbf{B}(x) dx \mathbf{u} - \int_0^L (\mathbf{N}(x))^\top q_x dx - (\mathbf{N}(L))^\top N_f - (\mathbf{N}(0))^\top N_i = \mathbf{0}$$



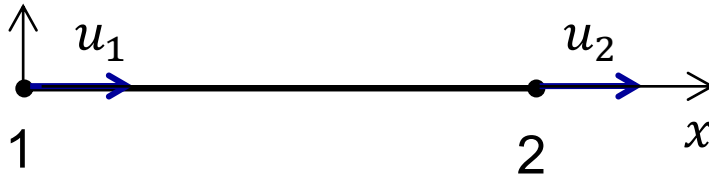
$\mathbf{k}$



$\mathbf{f}_e$



$$\mathbf{k} \mathbf{u} = \mathbf{f}_e$$



$$\mathbf{N}(x) = \begin{bmatrix} 1 - \frac{x}{L} & \frac{x}{L} \end{bmatrix}$$

$$\mathbf{B}(x) = \frac{1}{L} \begin{bmatrix} -1 & 1 \end{bmatrix}$$

Matriu de rigidesa:

$$\mathbf{k} = \frac{EA}{L} \int_0^L \begin{bmatrix} -1 \\ 1 \end{bmatrix} \begin{bmatrix} -1 & 1 \end{bmatrix} dx \quad \rightarrow \quad \mathbf{k} = \frac{EA}{L^2} \int_0^L \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} dx \quad \rightarrow \quad \mathbf{k} = \frac{EA}{L^2} \begin{bmatrix} L & -L \\ -L & L \end{bmatrix} \quad \rightarrow \quad \mathbf{k} = \frac{EA}{L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix}$$

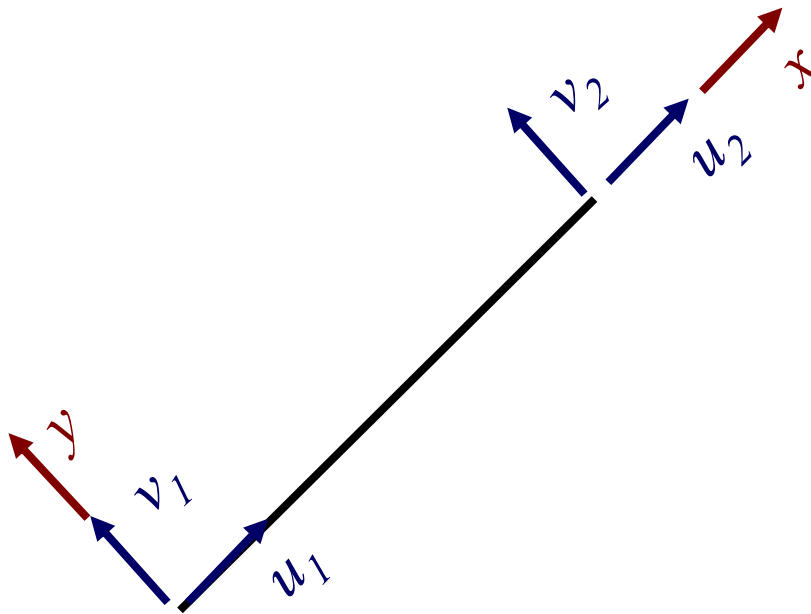
Vector de forces nodals:

$$\mathbf{f}_e = \int_0^L \begin{bmatrix} 1 - \frac{x}{L} \\ \frac{x}{L} \end{bmatrix} q_x dx + \begin{bmatrix} 0 \\ 1 \end{bmatrix} N_f + \begin{bmatrix} 1 \\ 0 \end{bmatrix} N_i \quad \rightarrow \quad \mathbf{f}_e = \int_0^L \begin{bmatrix} 1 - \frac{x}{L} \\ \frac{x}{L} \end{bmatrix} q_x dx + \begin{bmatrix} N_i \\ N_f \end{bmatrix}$$

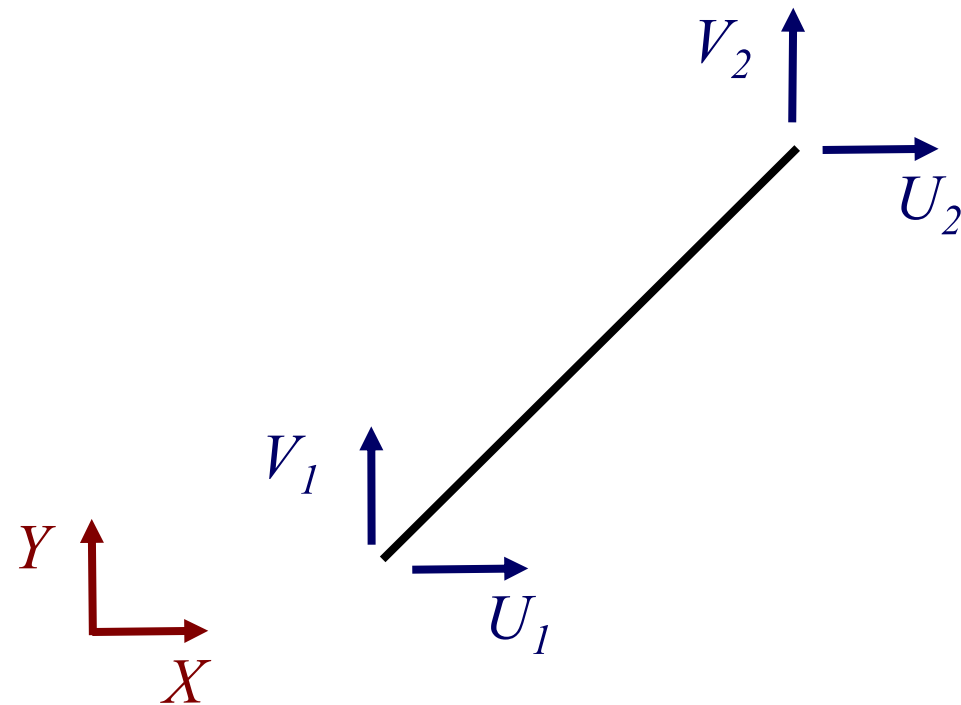
Elements de barra

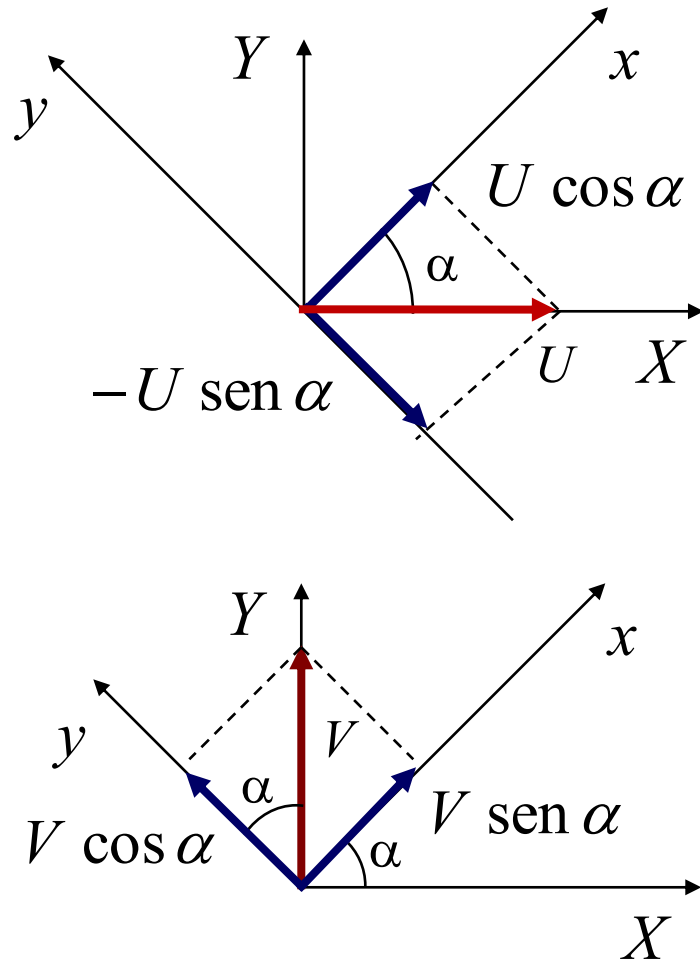
Transformació entre sistemes de coordenades

Sistema de coordenades local



Sistema de coordenades global





Pels desplaçaments

$$u = U \cos \alpha + V \sin \alpha$$

$$v = -U \sin \alpha + V \cos \alpha$$

$$\begin{Bmatrix} u \\ v \end{Bmatrix} = \begin{bmatrix} \cos \alpha & \sin \alpha \\ -\sin \alpha & \cos \alpha \end{bmatrix} \begin{Bmatrix} U \\ V \end{Bmatrix}$$

sistema
local

Matriu de
rotació

sistema
global

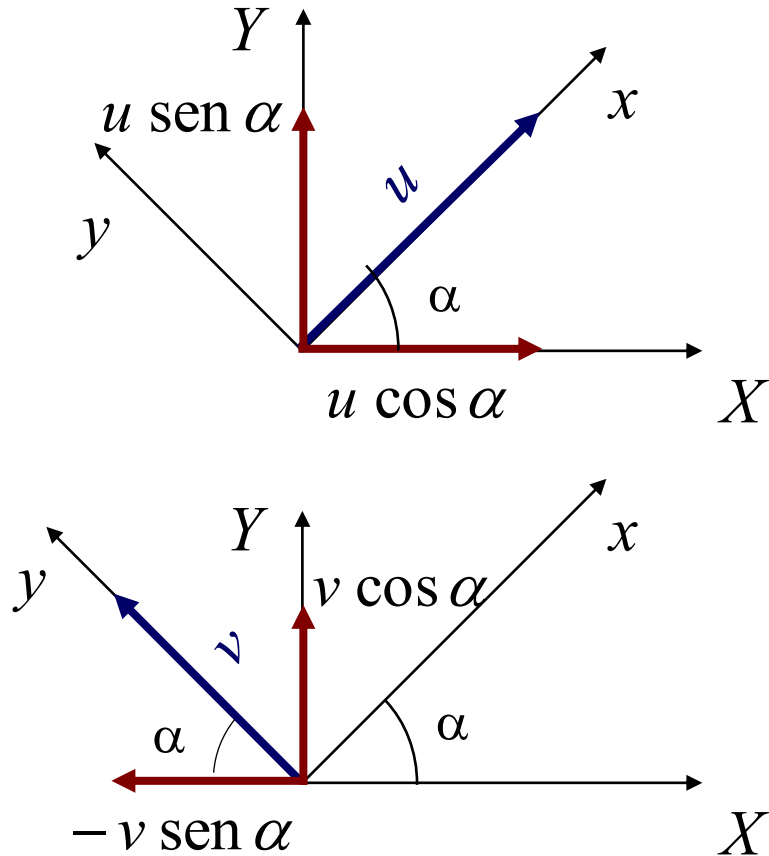
$$\mathbf{u} = \mathbf{R} \mathbf{U}$$

Per les forces:

$$f_x = F_x \cos \alpha + F_y \sin \alpha$$

$$f_y = -F_x \sin \alpha + F_y \cos \alpha$$

$$\mathbf{f} = \mathbf{R} \mathbf{F}$$



Pels desplaçaments

$$U = u \cos \alpha - v \sin \alpha$$

$$V = u \sin \alpha + v \cos \alpha$$

$$\begin{Bmatrix} U \\ V \end{Bmatrix} = \begin{bmatrix} \cos \alpha & -\sin \alpha \\ \sin \alpha & \cos \alpha \end{bmatrix} \begin{Bmatrix} u \\ v \end{Bmatrix}$$

$$\mathbf{U} = \mathbf{R}^T \mathbf{u}$$

Matriu $\mathbf{R}$ és
ortogonal

Per les forces:

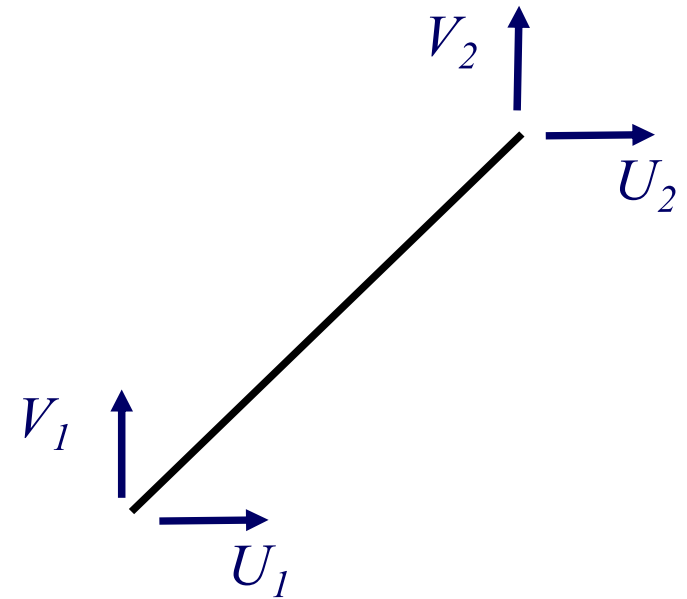
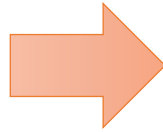
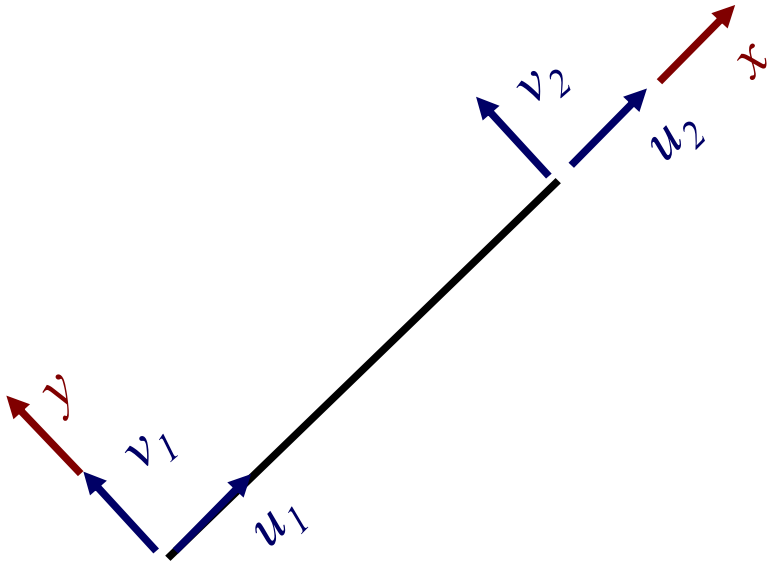
$$F_x = f_x \cos \alpha - f_y \sin \alpha$$

$$F_y = f_x \sin \alpha + f_y \cos \alpha$$

$$\mathbf{F} = \mathbf{R}^T \mathbf{f}$$

$$\begin{Bmatrix} u_1 \\ v_1 \\ u_2 \\ v_2 \end{Bmatrix} = \begin{bmatrix} \cos\alpha & \sin\alpha & 0 & 0 \\ -\sin\alpha & \cos\alpha & 0 & 0 \\ 0 & 0 & \cos\alpha & \sin\alpha \\ 0 & 0 & -\sin\alpha & \cos\alpha \end{bmatrix} \begin{Bmatrix} U_1 \\ V_1 \\ U_2 \\ V_2 \end{Bmatrix}$$

$$\mathbf{u} = \mathbf{R} \mathbf{U}$$





$$\mathbf{F}^{el} = \mathbf{K}^{el} \mathbf{U}^{el}$$

$\mathbf{K}^{el} ?$



$$\left\{ \begin{array}{l} \mathbf{u}^{el} = \mathbf{R}^{el} \mathbf{U}^{el} \\ \mathbf{f}^{el} = \mathbf{k}^{el} \mathbf{u}^{el} \end{array} \right.$$

$$\mathbf{f}^{el} = \mathbf{k}^{el} \mathbf{R}^{el} \mathbf{U}^{el}$$

$$\mathbf{R}^{el T} \mathbf{f}^{el} = \mathbf{R}^{el T} \mathbf{k}^{el} \mathbf{R}^{el} \mathbf{U}^{el}$$

$$\mathbf{R}^{el T} \mathbf{f}^{el} = \mathbf{R}^{el T} \mathbf{k}^{el} \mathbf{R}^{el} \mathbf{U}^{el}$$

$$\mathbf{F}^{el} = \mathbf{R}^{el T} \mathbf{f}^{el}$$

$$\mathbf{F}^{el} = \mathbf{R}^{el T} \mathbf{k}^{el} \mathbf{R}^{el} \mathbf{U}^{el}$$

$$\mathbf{F}^{el} = \mathbf{K}^{el} \mathbf{U}^{el}$$

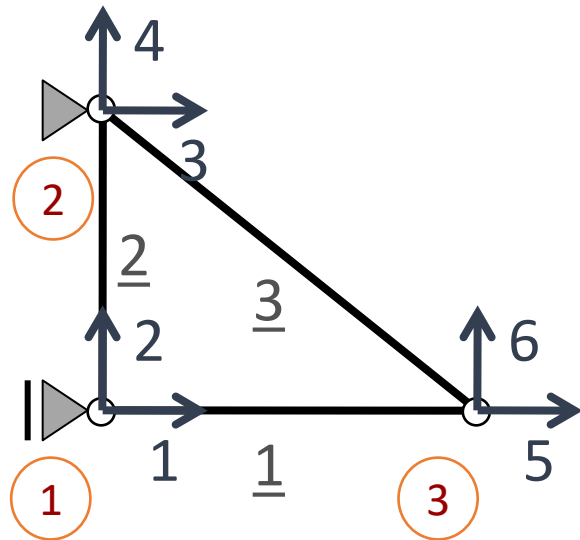
$$\mathbf{K}^{el} = \mathbf{R}^{el T} \mathbf{k}^{el} \mathbf{R}^{el}$$



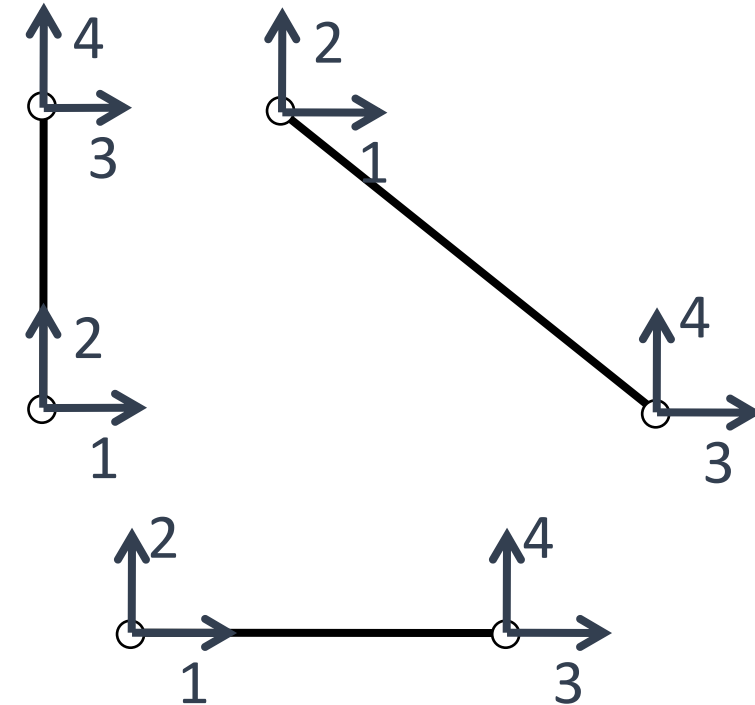
Elements de barra

Muntatge de la matriu de rigidesa

Matriu de rigidesa de l'estructura

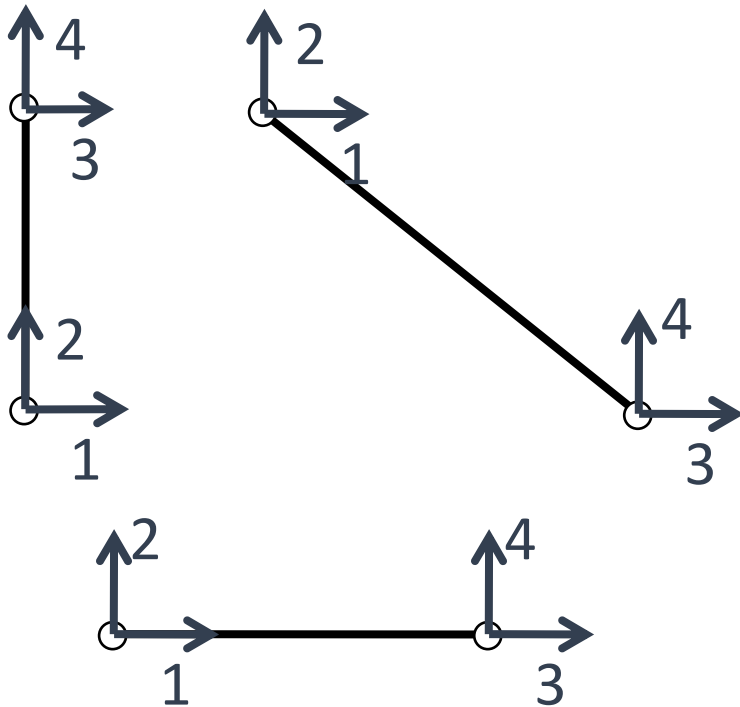


Estructura



Elements

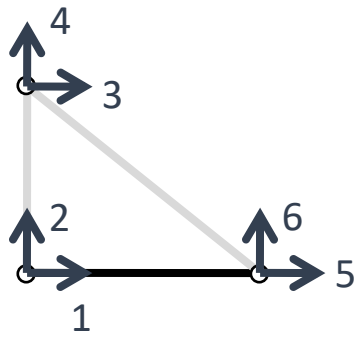
Matriu de rigidesa de l'element (sistema global)



	1	2	3	4
1	K_{11}^e	K_{12}^e	K_{13}^e	K_{14}^e
2	K_{21}^e	K_{22}^e	K_{23}^e	K_{24}^e
3	K_{31}^e	K_{32}^e	K_{33}^e	K_{34}^e
4	K_{41}^e	K_{42}^e	K_{43}^e	K_{44}^e

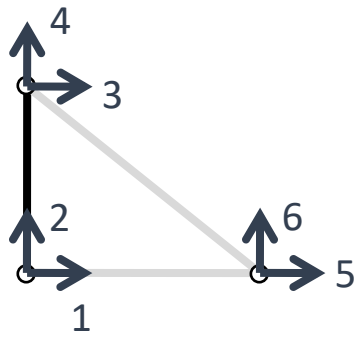
Matriu de rigidesa de l'estructura

	1	2	3	4	5	6
1	0	0	0	0	0	0
2	0	0	0	0	0	0
3	0	0	0	0	0	0
4	0	0	0	0	0	0
5	0	0	0	0	0	0
6	0	0	0	0	0	0



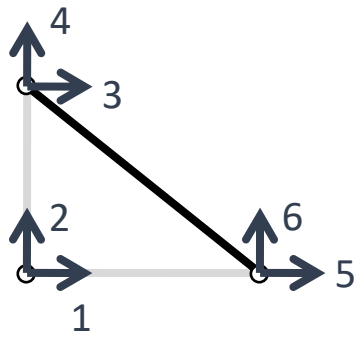
Element 1	1	2	3	4
				
Estructura	1	2	5	6

	1	2	3	4	5	6
1	K_{11}^{e1}	K_{12}^{e1}			K_{13}^{e1}	K_{14}^{e1}
2	K_{21}^{e1}	K_{22}^{e1}			K_{23}^{e1}	K_{24}^{e1}
3						
4						
5	K_{31}^{e1}	K_{32}^{e1}			K_{33}^{e1}	K_{34}^{e1}
6	K_{41}^{e1}	K_{42}^{e1}			K_{43}^{e1}	K_{44}^{e1}



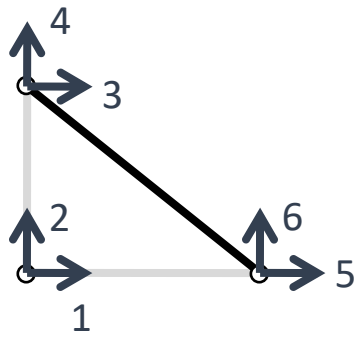
Element 2	1	2	3	4
				
Estructura	1	2	3	4

	1	2	3	4	5	6
1	$K_{11}^{e1} + K_{11}^{e2}$	$K_{12}^{e1} + K_{12}^{e2}$	K_{13}^{e2}	K_{14}^{e2}	K_{13}^{e1}	K_{14}^{e1}
2	$K_{21}^{e1} + K_{21}^{e2}$	$K_{22}^{e1} + K_{22}^{e2}$	K_{23}^{e2}	K_{24}^{e2}	K_{23}^{e1}	K_{24}^{e1}
3	K_{31}^{e2}	K_{32}^{e2}	K_{33}^{e2}	K_{34}^{e2}		
4	K_{41}^{e2}	K_{42}^{e2}	K_{43}^{e2}	K_{44}^{e2}		
5	K_{31}^{e1}	K_{32}^{e1}			K_{33}^{e1}	K_{34}^{e1}
6	K_{41}^{e1}	K_{42}^{e1}			K_{43}^{e1}	K_{44}^{e1}



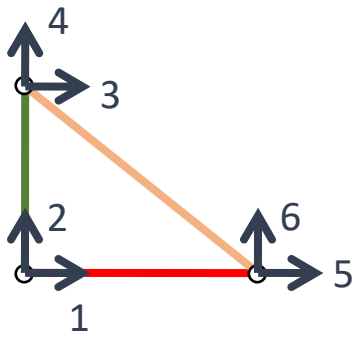
Element 3	1	2	3	4
Estructura	3	4	5	6

	1	2	3	4	5	6
1	$K_{11}^{e1} + K_{11}^{e2}$	$K_{12}^{e1} + K_{12}^{e2}$	K_{13}^{e2}	K_{14}^{e2}	K_{13}^{e1}	K_{14}^{e1}
2	$K_{21}^{e1} + K_{21}^{e2}$	$K_{22}^{e1} + K_{22}^{e2}$	K_{23}^{e2}	K_{24}^{e2}	K_{23}^{e1}	K_{24}^{e1}
3	K_{31}^{e2}	K_{32}^{e2}	$K_{33}^{e2} + K_{11}^{e3}$	$K_{34}^{e2} + K_{12}^{e3}$	K_{13}^{e3}	K_{14}^{e3}
4	K_{41}^{e2}	K_{42}^{e2}	$K_{43}^{e2} + K_{21}^{e3}$	$K_{44}^{e2} + K_{22}^{e3}$	K_{23}^{e3}	K_{24}^{e3}
5	K_{31}^{e1}	K_{32}^{e1}	K_{31}^{e3}	K_{32}^{e3}	$K_{33}^{e1} + K_{33}^{e3}$	$K_{34}^{e1} + K_{34}^{e3}$
6	K_{41}^{e1}	K_{42}^{e1}	K_{41}^{e3}	K_{42}^{e3}	$K_{43}^{e1} + K_{43}^{e3}$	$K_{44}^{e1} + K_{44}^{e3}$



Element 3	1	2	3	4
				
Estructura	3	4	5	6

	1	2	3	4	5	6
1	$K_{11}^{e1} + K_{11}^{e2}$	$K_{12}^{e1} + K_{12}^{e2}$	K_{13}^{e2}	K_{14}^{e2}	K_{13}^{e1}	K_{14}^{e1}
2	$K_{21}^{e1} + K_{21}^{e2}$	$K_{22}^{e1} + K_{22}^{e2}$	K_{23}^{e2}	K_{24}^{e2}	K_{23}^{e1}	K_{24}^{e1}
3	K_{31}^{e2}	K_{32}^{e2}	$K_{33}^{e2} + K_{11}^{e3}$	$K_{34}^{e2} + K_{12}^{e3}$	K_{13}^{e3}	K_{14}^{e3}
4	K_{41}^{e2}	K_{42}^{e2}	$K_{43}^{e2} + K_{21}^{e3}$	$K_{44}^{e2} + K_{22}^{e3}$	K_{23}^{e3}	K_{24}^{e3}
5	K_{31}^{e1}	K_{32}^{e1}	K_{31}^{e3}	K_{32}^{e3}	$K_{33}^{e1} + K_{33}^{e3}$	$K_{34}^{e1} + K_{34}^{e3}$
6	K_{41}^{e1}	K_{42}^{e1}	K_{41}^{e3}	K_{42}^{e3}	$K_{43}^{e1} + K_{43}^{e3}$	$K_{44}^{e1} + K_{44}^{e3}$



	1	2	3	4	5	6
1	$K_{11}^{e1} + K_{11}^{e2}$	$K_{12}^{e1} + K_{12}^{e2}$	K_{13}^{e2}	K_{14}^{e2}	K_{13}^{e1}	K_{14}^{e1}
2	$K_{21}^{e1} + K_{21}^{e2}$	$K_{22}^{e1} + K_{22}^{e2}$	K_{23}^{e2}	K_{24}^{e2}	K_{23}^{e1}	K_{24}^{e1}
3	K_{31}^{e2}	K_{32}^{e2}	$K_{33}^{e2} + K_{11}^{e3}$	$K_{34}^{e2} + K_{12}^{e3}$	K_{13}^{e3}	K_{14}^{e3}
4	K_{41}^{e2}	K_{42}^{e2}	$K_{43}^{e2} + K_{21}^{e3}$	$K_{44}^{e2} + K_{22}^{e3}$	K_{23}^{e3}	K_{24}^{e3}
5	K_{31}^{e1}	K_{32}^{e1}	K_{31}^{e3}	K_{32}^{e3}	$K_{33}^{e1} + K_{33}^{e3}$	$K_{34}^{e1} + K_{34}^{e3}$
6	K_{41}^{e1}	K_{42}^{e1}	K_{41}^{e3}	K_{42}^{e3}	$K_{43}^{e1} + K_{43}^{e3}$	$K_{44}^{e1} + K_{44}^{e3}$